

# Spin Stability of Torque-Free Systems—Part II

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Part I of this paper contains the development of a method for constructing stability criteria applicable to spinning motions of torque-free, elastic, dissipative systems possessing a finite number of degrees of freedom. In Part II, this method is applied to a spinning, rigid satellite that carries four elastically mounted antennas.

## Introduction

THE concluding section of Part I contains a description of a procedure for the formulation of stability conditions. This procedure consists of six steps, designated (1–6), the first five of which suffice for the purpose of establishing stability. These five steps, numbered as such, are carried out in the sequel.

## Analysis

1) Figure 1 shows a system  $S$  consisting of a "main body."  $B$ , and four "antennas"  $A_r$  ( $r = 1, \dots, 4$ ).  $B$  is a rigid body having three (in general) unequal central principal moments of inertia.  $A_r$  ( $r = 1, \dots, 4$ ) are "slender" rigid bodies. That is,  $A_r$  possesses a "central axis,"  $L_r$ , such that the mass center,  $A_r^*$ , of  $A_r$  lies on  $L_r$ ; the moments of inertia of  $A_r$  about all lines passing through  $A_r^*$  and normal to  $L_r$  are equal to each other; and the moment of inertia of  $A_r$  about  $L_r$  is negligible.

$A_r$  ( $r = 1, \dots, 4$ ) are attached to  $B$  at points  $P_r$  ( $r = 1, \dots, 4$ ) whose coordinates relative to  $X_1, X_2, X_3$ , the principal axes of inertia of  $B$  for the mass center  $B^*$  of  $B$ , are respectively  $(R_1, 0, R_3)$ ,  $(R_1, R_2, 0)$ ,  $(R_1, 0, -R_3)$ ,  $(R_1, -R_2, 0)$ . The connection between  $A_r$  and  $B$  is effected by means of a revolute joint, a linear torsional spring, and a linear, viscous, torsional damper. The orientations of the axes of the joints, the spring constants, and the damping constants associated with these connections are given in Table 1, as are the distance from  $P_r$  to  $A_r^*$ , the mass of  $A_r$ , and the moment of inertia of  $A_r$  about any line passing through  $A_r^*$  and normal to  $L_r$ . Body  $B$  has a mass  $m$  and a moment of inertia  $B_i$  about  $X_i$  ( $i = 1, 2, 3$ ).

The system has four internal degrees of freedom. As we shall be concerned with motions during which the angles between  $L_r$  ( $r = 1, \dots, 4$ ) and  $X_1$  remain constant, we use as generalized coordinates,  $q_r$  ( $r = 1, \dots, 4$ ), the radian measures of the deviations of the angles between  $L_r$  and  $X_1$  during a general motion from the constants values of interest, denoting the latter by  $\alpha$  in the case of  $A_1$  and  $A_3$ , and by  $\beta$  for  $A_2$  and  $A_4$ , as indicated in Fig. 1. If the springs at  $P_r$  ( $r = 1, \dots, 4$ ) are undeformed when  $\alpha + q_r$  ( $r = 1, 3$ ) and  $\beta + q_r$  ( $r = 2, 4$ ) are equal to zero, the system of forces transmitted by  $B$  to  $A_r$  is then equivalent to a force whose line of action passes through  $P_r$ , together with a couple whose torque,  $T_r$  ( $r = 1, \dots, 4$ ), is given by

$$T_1 = -[k_1(\alpha + q_1) + \mu_1 \dot{q}_1]b_2 \quad (1a)$$

$$T_2 = [k_2(\beta + q_2) + \mu_2 \dot{q}_2]b_3 \quad (1b)$$

Table 1 System parameters

	$r = 1$	$r = 2$	$r = 3$	$r = 4$
Orientation of joint axis at $P_r$	$X_2$	$X_3$	$X_2$	$X_3$
Spring constant at $P_r$	$k_1$	$k_2$	$k_1$	$k_2$
Damping constant at $P_r$	$\mu_1$	$\mu_2$	$\mu_1$	$\mu_2$
Distance from $P_r$ to $A_r^*$	$a$	$b$	$a$	$b$
Mass of $A_r$	$M$	$N$	$M$	$N$
Transverse moment of inertia of $A_r$	$I$	$J$	$I$	$J$

$$T_3 = [k_1(\alpha + q_3) + \mu_1 \dot{q}_3]b_2 \quad (1c)$$

$$T_4 = -[k_2(\beta + q_4) + \mu_2 \dot{q}_4]b_3 \quad (1d)$$

where  $b_i$  ( $i = 1, 2, 3$ ) are unit vectors directed as shown in Fig. 1. The corresponding generalized forces are

$$F_{q_r} = -[k_1(\alpha + q_r) + \mu_1 \dot{q}_r] \quad r = 1, 3 \quad (2a)$$

$$F_{q_r} = -[k_2(\beta + q_r) + \mu_2 \dot{q}_r] \quad r = 2, 4 \quad (2b)$$

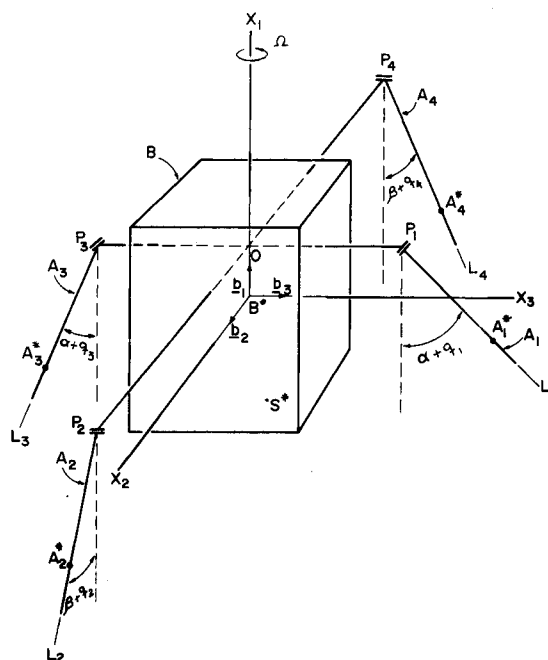


Fig. 1 Four antennae system.

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The requirements imposed by Eqs. [(2-4) Part I] can, therefore, be satisfied by defining  $V$  and  $D_r$  as

$$V \triangleq \frac{1}{2}k_1[(\alpha + q_1)^2 + (\alpha + q_3)^2] + \frac{1}{2}k_2[(\beta + q_2)^2 + (\beta + q_4)^2] \quad (3)$$

and

$$D_r \triangleq -\mu_1 \dot{q}_r \quad r = 1, 3, \quad D_r \triangleq -\mu_2 \dot{q}_r \quad r = 2, 4 \quad (4)$$

2) To describe a simple spin which proceeds with a constant angular speed  $\Omega$  and throughout which the angular momentum of  $S$  relative to the mass center,  $S^*$ , of  $S$  has a constant magnitude  $H$ , we let  $\omega$  denote the angular velocity of  $B$  in a newtonian reference frame and, after defining  $\omega_i$  ( $i = 1, 2, 3$ ) as

$$\omega_i \triangleq \omega \cdot \mathbf{b}_i \quad i = 1, 2, 3 \quad (5)$$

take

$$\omega_1 = \Omega, \quad \omega_2 = \omega_3 = 0, \quad q_r = 0 \quad r = 1, \dots, 4 \quad (6)$$

For  $Z_1, Z_2, Z_3$ , we take axes originating at  $S^*$  and respectively parallel to  $X_1, X_2, X_3$ , since the latter axes are principal axes of  $S$  for  $S^*$  when  $q_r = 0$  ( $r = 1, \dots, 4$ ). Next, direction cosine matrices and position vector components suitable for use in Eqs. [(79-81) Part I] can be formulated for each body, in which connection it is helpful to introduce the following abbreviations:

$$s_r \triangleq \begin{cases} \sin(\alpha + q_r) & r = 1, 3 \\ \sin(\beta + q_r) & r = 2, 4 \end{cases} \quad (7a)$$

$$c_r \triangleq \begin{cases} \cos(\alpha + q_r) & r = 1, 3 \\ \cos(\beta + q_r) & r = 1, 4 \end{cases} \quad (7b)$$

$$Q \triangleq m + 2(M + N) \quad (8)$$

For  $B$

$$[C_{jk}^B] = \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} \quad (9a)$$

$$p_1^B = -R_1 + Q^{-1}[mR_1 + aM(c_1 + c_3) + bN(c_2 + c_4)] \quad (9b)$$

$$p_2^B = -aMQ^{-1}(s_1 - s_3), \quad p_3^B = -bNQ^{-1}(s_2 - s_4) \quad (9c)$$

while for  $A_1$

$$[C_{jk}^{A_1}] = \begin{bmatrix} c_1 & 0 & s_1 \\ 0 & 1 & 0 \\ -s_1 & 0 & c_1 \end{bmatrix} \quad (10a)$$

$$p_1^{A_1} = -bc_1 + Q^{-1}[mR_1 + aM(c_1 + c_3) + bN(c_2 + c_4)] \quad (10b)$$

$$p_2^{A_1} = -aMQ^{-1}(s_1 - s_3), \quad p_3^{A_1} = -bNQ^{-1}(s_2 - s_4) + R_3 + as_1 \quad (10c)$$

and similarly for  $A_2, \dots, A_4$ . Using Eq. (79) (Part I), and letting

$$s\alpha \triangleq \sin \alpha, \quad c\alpha \triangleq \cos \alpha, \quad s\beta \triangleq \sin \beta, \quad c\beta \triangleq \cos \beta \quad (11)$$

one thus arrives at

$$\tilde{I}_{11} = B_1 + 2[I s^2 \alpha + J s^2 \beta + M(R_3 + as\alpha)^2 + N(R_2 + bs\beta)^2] \quad (12a)$$

$$\tilde{I}_{22} = B_2 + 2\{I + Jc^2 \beta + N[R_1 mQ^{-1} - b(1 - 2NQ^{-1})c\beta + 2aMQ^{-1}c\alpha]^2 + M\{[R_1 mQ^{-1} - a(1 - 2MQ^{-1})c\alpha + 2bNQ^{-1}c\beta]^2 + (R_3 + as\alpha)^2\} \quad (12b)$$

$$\tilde{I}_{33} = B_3 + 2\{J + Ic^2 \alpha + M[R_1 mQ^{-1} - a(1 - 2MQ^{-1})c\alpha + 2bNQ^{-1}c\beta]^2 + N\{[R_1 mQ^{-1} - b(1 - 2NQ^{-1})c\beta + 2aMQ^{-1}c\alpha]^2 + (R_2 + bs\beta)^2\} \quad (12c)$$

while Eqs. (80) (Part I) furnish

$$\tilde{I}_{11,1} = \tilde{I}_{11,3} = 2[(I + a^2 M)s\alpha + aR_3 M c\alpha] \quad (13a)$$

$$\tilde{I}_{11,2} = \tilde{I}_{11,4} = 2[(J + b^2 N)s\beta + bR_2 N c\beta] \quad (13b)$$

$$\tilde{I}_{12,1} = \tilde{I}_{12,3} = \tilde{I}_{13,2} = \tilde{I}_{13,4} = 0 \quad (13c)$$

$$\tilde{I}_{12,2} = -\tilde{I}_{12,4} = Jc2\beta - N\{bs\beta(R_2 + bs\beta) + R_1 mQ^{-1} - b(1 - 2NQ^{-1})c\beta + 2aMQ^{-1}c\alpha\}b(1 - 2NQ^{-1})c\beta - 2bNQ^{-2}[R_1 mN + 2aMNc\alpha - bN(2M + m)c\beta]c\alpha \quad (13d)$$

$$\tilde{I}_{13,1} = -\tilde{I}_{13,3} = Ic2\alpha - M\{as\alpha(R_3 + as\alpha) + [R_1 mQ^{-1} - a(1 - 2MQ^{-1})c\alpha + 2bNQ^{-1}c\beta]a(1 - 2MQ^{-1})c\alpha\} - 2aMQ^{-2}[R_1 mM + 2bMNc\beta - aM(2N + m)c\alpha]c\beta \quad (13e)$$

and Eqs. (81) (Part I) yield

$$\tilde{I}_{11,12} = \tilde{I}_{11,23} = \tilde{I}_{11,34} = \tilde{I}_{11,41} = 0 \quad (14a)$$

$$\tilde{I}_{11,11} = \tilde{I}_{11,33} = 2[Ic2\alpha - aM(R_3 + as\alpha)s\alpha + a^2 M(1 - MQ^{-1})c^2 \alpha] \quad (14b)$$

$$\tilde{I}_{11,22} = \tilde{I}_{11,44} = 2[Jc2\beta - bN(R_2 + bs\beta)s\beta + b^2 N(1 - NQ^{-1})c^2 \beta] \quad (14c)$$

$$\tilde{I}_{11,33} = 2a^2 M^2 Q^{-1} c^2 \alpha, \quad \tilde{I}_{11,24} = 2b^2 N^2 Q^{-1} c^2 \beta \quad (14d)$$

3) To determine the circumstances under which the simple spin described by Eq. (6) can occur, we note that, from Eq. (3)

$$\tilde{V}_{,1} = \tilde{V}_{,3} = k_1 \alpha, \quad \tilde{V}_{,2} = \tilde{V}_{,4} = k_2 \beta \quad (15)$$

Using these results, Eqs. (13), and

$$H^2 = (\tilde{I}_{11} \Omega)^2 \quad (16)$$

in Eq. (84) (Part I) with  $r = 1$  or  $r = 3$ , we then obtain

$$k_1 = \Omega^2 \alpha^{-1} [Is\alpha c\alpha + aM c\alpha(R_3 + as\alpha)] \quad (17a)$$

while for  $r = 2$  or  $r = 4$ ,

$$k_2 = \Omega^2 \beta^{-1} [Js\beta c\beta + bN c\beta(R_2 + bs\beta)] \quad (17b)$$

The simple spin under consideration thus can occur only when these equations are satisfied. In this connection it should be kept in mind that  $k_1$  and  $k_2$  are intrinsically positive.

We note for future reference that, from Eq. (3),

$$\tilde{V}_{,11} = \tilde{V}_{,33} = k_1, \quad \tilde{V}_{,22} = \tilde{V}_{,44} = k_2 \quad (18a)$$

$$\tilde{V}_{,rs} = 0 \quad \text{if} \quad r \neq s \quad (18b)$$

4) The external stability conditions [see the inequalities (82) (Part I)]

$$\tilde{I}_{11} - \tilde{I}_{22} > 0, \quad \tilde{I}_{11} - \tilde{I}_{33} > 0 \quad (19)$$

can now be expressed in terms of system parameters by substitution from Eq. (12).

5) All ingredients required for the evaluation of  $\tilde{Z}_{rs}$  [see Eq. (83) Part I] are available in Eqs. (12-18); and, if  $\Delta_r$  ( $r = 1, \dots, 4$ ) are defined as

$$\Delta_1 \triangleq \tilde{Z}_{,11}, \quad \Delta_2 \triangleq \begin{vmatrix} \tilde{Z}_{,11} & \tilde{Z}_{,12} \\ \tilde{Z}_{,21} & \tilde{Z}_{,22} \end{vmatrix} \quad (20a)$$

$$\Delta_3 \triangleq \begin{vmatrix} \tilde{Z}_{,11} & \tilde{Z}_{,12} & \tilde{Z}_{,13} \\ \tilde{Z}_{,21} & \tilde{Z}_{,22} & \tilde{Z}_{,23} \\ \tilde{Z}_{,31} & \tilde{Z}_{,32} & \tilde{Z}_{,33} \end{vmatrix}, \quad \Delta_4 \triangleq \begin{vmatrix} \tilde{Z}_{,11} & \tilde{Z}_{,12} & \tilde{Z}_{,13} & \tilde{Z}_{,14} \\ \tilde{Z}_{,21} & \tilde{Z}_{,22} & \tilde{Z}_{,23} & \tilde{Z}_{,24} \\ \tilde{Z}_{,31} & \tilde{Z}_{,32} & \tilde{Z}_{,33} & \tilde{Z}_{,34} \\ \tilde{Z}_{,41} & \tilde{Z}_{,42} & \tilde{Z}_{,43} & \tilde{Z}_{,44} \end{vmatrix} \quad (20b)$$

then, in accordance with Sylvester's criteria, the internal stability conditions [see the inequalities (63) (Part I)] can be stated as

$$\Delta_r > 0 \quad (r = 1, \dots, 4) \quad (21)$$

Hence, given the values of the "rest" angles  $\alpha$  and  $\beta$ , the inertia parameters  $m, B_1, B_2, B_3$  for body  $B$ , the hinge point location parameters  $R_1, R_2, R_3$ , and the inertia parameters  $a, b, M, N, I, J$  for the antennae  $A_r$  ( $r = 1, \dots, 4$ ), one can now establish stability of the simple spin under consideration by means of elementary calculations. For example, suppose  $\alpha = \beta = 60^\circ$ ;  $m = 500, B_1 = 110, B_2 = 100, B_3 = 70; R_1 = 0.5, R_2 = R_3 = 0.3$ ; and  $a = b = 0.2, M = N = 2, I = J = 0.02$ . Here, as in the sequel, all lengths, masses, and moments of inertia are expressed in units of meters, kilograms, and kilogram-(meters)<sup>2</sup>, respectively. Then, for instance,

$$\tilde{I}_{11} = 111.8, \quad \tilde{I}_{11,2} = 0.207, \quad \tilde{I}_{11,22} = -0.308 \quad (12a) \quad (13b) \quad (14c)$$

$$k_1 = 0.09864\Omega^2, \quad \tilde{V}_{,2} = 0.1033\Omega^2, \quad \tilde{V}_{,44} = 0.0986\Omega^2 \quad (17a) \quad (15) \quad (18a)$$

$$H^2 = 12511 \cdot \Omega^2, \quad \tilde{Z}_{,33} = 0.2514\Omega^2 \quad (16) \quad (9.2)$$

From the inequalities (19), the external stability conditions are

$$9.646 > 0, \quad 39.65 > 0$$

and these are clearly satisfied. And from Eq. (20),

$$\Delta_1 = 0.2514\Omega^2, \quad \Delta_2 = 0.06195\Omega^4$$

$$\Delta_3 = 0.01557\Omega^6, \quad \Delta_4 = 0.003835\Omega^8$$

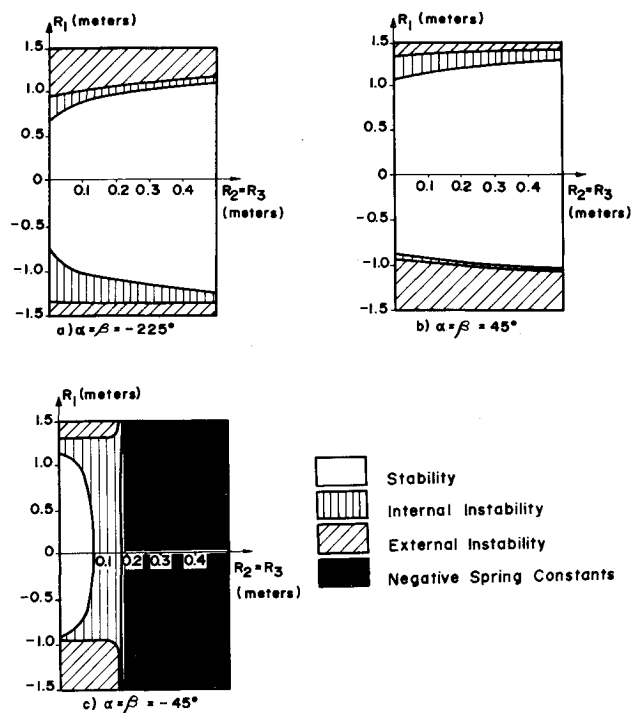


Fig. 2 Four antennae system stability charts.

Hence the internal stability conditions [the inequalities (21)] also are satisfied, and we are dealing with a stable simple spin.

The method at hand is particularly well suited for dealing with questions of the kind encountered in preliminary design studies. For example, one may wish to explore alternative schemes for the deployment of the antennae, given the inertia properties of

both the main body and the antennae. Suppose, for instance, that with  $m, B_1, B_2, B_3, a, b, M, N, I$ , and  $J$  as before one wished to place the points  $P_r$  ( $r=1, \dots, 4$ ) (see Fig. 1) in such a way that  $\alpha = \beta = -225^\circ$  be permissible. (This sort of configuration is attractive because it permits use of the torsion springs at  $P_r$  ( $r=1, \dots, 4$ ) for two purposes: pressing the antennae against the main body during "spin up," that is, while  $\Omega$  is made to increase from zero to its design value; and holding the antennae in position when  $\Omega$  has attained the design value.) Figure 2a shows a stability chart obtained by using the inequalities (19) and (20) with the given parameter values for various values of  $R_1$  and  $R_2 = R_3$ . Horizontal shading indicates that at least one of the inequalities (19) is violated; vertical shading corresponds to failure to satisfy all of the inequalities (21); and for each point of the unshaded region simple spin about  $X_1$  is a stable motion. With this chart in hand, one can place  $P_r$  ( $r=1, \dots, 4$ ) in an infinite number of permissible ways; and the value of  $\Omega$  associated with a particular placement of  $P_r$  ( $r=1, \dots, 4$ ) can be assigned at will, provided  $k_1$  and  $k_2$  be chosen in accordance with Eq. (17).

Questions regarding the relative merits of various antennae arrangements fall outside of the limits of the present work. However, it is worth mentioning in conclusion that the system under consideration offers a considerable variety of possible arrangements. This is apparent from Fig. 2, which indicates that there exist stable spins about  $X_1$  in the following ranges:  $-270^\circ < \alpha = \beta \leq -180^\circ$  (Fig. 2a),  $0^\circ < \alpha = \beta \leq 90^\circ$  (Fig. 2b), and  $-90^\circ \leq \alpha = \beta < 0^\circ$  (Fig. 2c). (For the parameter values used to construct these stability charts, and in the given ranges for  $R_1$  and  $R_2 = R_3$ , we have found no stable spins about  $X_1$  for  $-90^\circ < \alpha = \beta < 0^\circ$ . The solidly black portion in Fig. 2c corresponds to configurations that must be ruled out because the associated values of  $k_1$  and  $k_2$  are negative. In any event, this chart deals with a situation that is of more theoretical than practical interest.) Finally, we should like to stress that it is not necessary to take  $\alpha = \beta, R_2 = R_3, a = b, M = N$ , and  $I = J$ . This was done in connection with Fig. 2 solely for the purpose of minimizing the complexity of an illustrative example.